

Stabilizing a CFD model of an unstable system through model reduction

S. Hovland and J. T. Gravdahl
Department of Engineering Cybernetics, NTNU

Abstract

We demonstrate stabilization of a computational fluid dynamics model of an unstable system. The unstable heating of a two-dimensional plate is used as a case study. Active control is introduced by cooling parts of the boundaries of the plate. The high order of the original model is reduced by proper orthogonal decomposition, giving an unstable reduced order model with a state space structure convenient for controller design. A stabilizing controller based on pole placement is designed for the reduced order model and integral action is included to enhance performance. The controller is then applied to the full model, where it is shown through simulations to stabilize the system. The demonstrated procedure makes it possible to analyze stability properties and design control systems for a class of systems that would otherwise be very computationally demanding.

1 Introduction

Control of fluid flow is a research area that has gained an increasing amount of interest over the last decade, with a recent contribution in [1]. Both internal flow (flow inside bounded regions, such as within engines or turbomachinery) and external flow (flow around vehicles or bodies) have recently been subject to an increasing amount of interest from the control community. A nice review of advances made in the intersection between control theory and fluid mechanics can be found in [2].

Distributed systems such as flow dynamics are modeled mathematically by a set of partial differential equations (PDEs). In order to simulate such systems on a computer, the PDEs are usually discretized using a set of tools known as computational fluid dynamics (CFD) ([3], [4], [5]). Although often acceptable for simulation purposes, the resulting discretized models are often of such

high order that it may be infeasible to use them in controller design. For instance, numerical simulation of the full three-dimensional Navier-Stokes equations is still too costly for the purpose of optimization and control of unsteady flows [6].

There is, however, a potential for using model order reduction tools to reduce the complexity of the models. Model reduction is a powerful tool, and a recent survey of different techniques may be found in [7]. Although many of them have been successfully used within the control community, methods such as Hankel model reduction and balanced truncation are too computationally demanding for systems of such high order as those encountered in many CFD applications.

Proper orthogonal decomposition (POD) is a popular technique that is still applicable even for very high-order systems, using the method of snapshots that will be further presented in section 2. Nice examples of this approach have been shown e.g. in [6] and [8], in active control of fluid flow over a backward-facing step and forward-facing step, respectively, using finite element discretization of the Navier-Stokes equations. In [9] and [10] it was shown that the reduced order models can have a state space structure, which is convenient for controller design. In this paper we extend the analysis of the unsteady two-dimensional heat equation studied in [10] to also include a heat source that renders the plate model unstable.

The paper is organized as follows. In section 2 the model order reduction technique is presented. The full CFD model of the unstable system is derived in section 3.1, and the reduced order model is found in section 3.2. A controller is designed in section 4 and validated in section 5, before concluding remarks are offered in section 6.

2 POD model order reduction

First introduced independently by Karhunen [11] and Loève [12], POD is sometimes called the Karhunen-Loève expansion. When first introduced in the context of fluid mechanics in [13], it was used to study turbulent flows. Applicable even for complex non-linear problems, POD makes it possible to keep the essential dynamics of the original model in order to perform the controller design. We will now outline the procedure.

Consider the general transient model of a physical system

$$\frac{\partial x}{\partial t} = \mathcal{D}(x), \quad (1)$$

where $x \in \mathbb{R}^n$ is the state, ξ is a spatial coordinate and $\mathcal{D}(x)$ is some nonlinear partial differential operator in the spatial variables [14]. By assuming that the state x can be represented as a linear combination of r basis functions,

$$x_r = \Phi_r \alpha = \sum_{i=1}^r \phi_i \alpha_i, \quad (2)$$

where $x_r \in \mathbb{R}^r$ is an approximation of the state, a reduced order model of (1) can be derived. The matrix $\Phi_r \in \mathbb{R}^{n \times r}$ is an orthogonal projection matrix (i.e. $\Phi_r^T \Phi_r = I$) whose columns are orthonormal basis vectors ϕ_i that span the ‘space’ of spatial dynamics of the state, and the α_i ’s are time varying expansion coefficients (also called modal amplitudes).

The POD offers a way to find a reduced set of basis functions ϕ_i , based on an ensemble of data of the state trajectories $x(t)$, using the method of snapshots [15]. The snapshots may be taken from physical experiments or from computer (CFD) simulations. Since the procedure is purely data-driven, i.e. the basis functions are found without considering the governing equations, finding the snapshot data is critical, since the reduced-order model will capture only the dynamics present in the data. A suitable input should therefore be used to excite the system, so that the desired characteristics are present in the data. How to best choose suitable inputs is discussed in [10].

The snapshots are collected in a matrix \mathcal{X} ,

$$\mathcal{X} = [x^1, x^2, \dots, x^N] = [x(t_1), x(t_2), \dots, x(t_N)], \quad (3)$$

where the columns $\{\mathcal{X}_j\}_{j=1}^N$ can be thought of as the spatial coordinate vectors of the system at

time step t_j . The rows $\{\mathcal{X}_i\}_{i=1}^n$ describe the time trajectories of the system evaluated at different locations in the spatial domain [16].

The POD basis vectors ϕ can be found by choosing the orthonormal set that maximizes the cost [17]

$$\max_{\phi} \frac{\langle |x(\varphi)|^2 \rangle}{(\varphi, \varphi)} = \frac{\langle |x(\phi)|^2 \rangle}{(\phi, \phi)},$$

where (\cdot, \cdot) is the scalar product operator and $\langle \cdot \rangle$ is a time averaging operator. Equivalently, the basis functions can be found by performing singular value decomposition (SVD) of the snapshot matrix \mathcal{X} , which is the procedure followed in this paper. A rigorous treatment of the connection between SVD and POD can be found in [18]. By SVD, the snapshot matrix is decomposed as

$$\mathcal{X} = \Phi \sum \Psi^T, \quad (4)$$

where the columns of $\Phi = [\Phi_1, \dots, \Phi_n]$ form the optimal orthogonal basis for the space spanned by \mathcal{X} , Φ and Ψ are unitary matrices (i.e. $\Phi^{-1} = \Phi^T, \Psi^{-1} = \Psi^T$) and \sum is a diagonal matrix with the singular values σ of \mathcal{X} on the diagonal. The r most significant basis functions are associated with the r largest singular values of \mathcal{X} . If the singular values fall off rapidly, a reduced order model may be constructed that captures the most salient characteristics of the original system.

The time varying expansion coefficients α_i in equation (2) are determined through Galerkin projection, a well-known method for generating a system of ordinary differential equations from a system of partial differential equations [19], [20]. To ensure that the original model is approximated as accurately as possible, α_i are found by requiring that the projection of the truncated solutions onto the basis is zero,

$$\left(\phi_i, \frac{\partial x_r}{\partial t} - \mathcal{D}(x_r) \right) = 0, \quad i = \{1, \dots, r\}. \quad (5)$$

By posing this constraint on $\alpha_i(t)$ for a given set of basis functions ϕ_i , the expansion coefficients α_i can be found by inserting (2) into (5) to yield r initial value problems for α :

$$\dot{\alpha}(t) = \left(\mathcal{D} \left(\sum_{i=1}^r \phi_i \alpha_i \right), \phi_i \right). \quad (6)$$

If the initial values of equation (1) are given as $x(0, \xi) = f(\xi)$, where ξ is a spatial variable, the initial value for α is found from

$$\alpha_i(0) = (\phi_i, f). \quad (7)$$

Thus the problem of solving a possibly complex PDE is reduced to that of solving a set of ordinary differential equations.

Eventually, pursuing a reduced order system on the regular linear state space form, the model reduction objective is to find a model

$$\begin{aligned}\hat{x}(k+1) &= \hat{A}\hat{x}(k) + \hat{B}u(k), \\ \hat{y}(k) &= \hat{C}\hat{x}(k),\end{aligned}\quad (8)$$

where $\hat{x} \in \mathbb{R}^r$, $\hat{A} \in \mathbb{R}^{r \times r}$, $u \in \mathbb{R}^m$, $\hat{B} \in \mathbb{R}^{r \times m}$, $y \in \mathbb{R}^s$ and $\hat{C} \in \mathbb{R}^{s \times r}$ where $r \ll n$, that approximates the system (1).

3 Case study: Heated plate

3.1 Full CFD model

To demonstrate how an unstable system can be stabilized using POD and feedback control, we study heat conduction in a plate. The plate is 1 m \times 1 m, 1 cm thick, defining the two-dimensional¹ computational domain $\Omega = [0, 1] \times [0, 1]$ depicted in figure 1. The plate is insulated along the boundaries, apart from the center of each boundary, where four flux actuators are located. This defines Neumann boundary conditions on all boundaries. The temperature $T(t, x, y)$ on

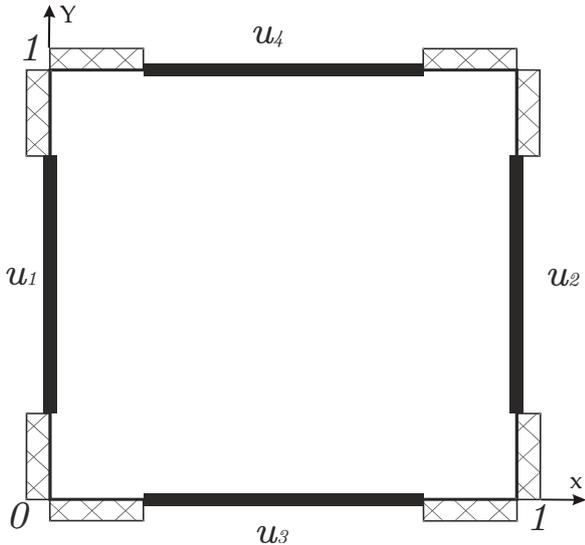


Figure 1: Sketch of plate with actuators on boundaries (bold lines).

¹A two-dimensional analysis is valid since the plate is so thin.

the plate is governed by the unsteady linear two-dimensional heat equation

$$\rho c_p \frac{\partial T}{\partial t} = k \frac{\partial^2 T}{\partial x^2} + k \frac{\partial^2 T}{\partial y^2} + S, \quad (9)$$

where ρ and c_p are the density and specific heat capacity of the plate, respectively, and k is the thermal conductivity, that is assumed to be uniform over the computational domain and independent of the temperature. Note that x now and in the following denotes a spatial coordinate and no longer the state variable. The term $S \triangleq S_c + S_T$ describes heat sinks and sources. In the present problem, convective heat transfer to the surroundings gives rise to a term

$$S_c = hA(T - T_\infty) \text{ [W]}, \quad (10)$$

where h is the convective heat transfer coefficient, A is the heat transfer area of the surface and T_∞ is the ambient temperature. Expression (10) is a source term whenever $T > T_\infty$, and a sink term whenever $T < T_\infty$. Due to electric current, the plate is subject to an internal temperature-dependent heat source

$$S_T = k_1 T \text{ [W/m}^3\text{]},$$

where $k_1 > 0$, at all points except from the boundary. Intuitively, this positive feedback from the temperature to the source may lead to a physically unstable system if the convective heat loss to the surroundings is not large enough. An increase in temperature will then lead to a stronger source, which again increases the temperature, and so on. Discretizing the governing equation by the finite volume method, (9) is integrated over each control volume CV and over the time interval from t to $t + \Delta t$, to obtain [5]

$$\begin{aligned}\int_{CV} \left(\int_t^{t+\Delta t} \rho c_p \frac{\partial T}{\partial t} dt \right) dV = \\ \int_t^{t+\Delta t} \int_{CV} \left(k \frac{\partial^2 T}{\partial x^2} \right) dV dt \\ + \int_t^{t+\Delta t} \int_{CV} \left(k \frac{\partial^2 T}{\partial y^2} \right) dV dt \\ + \int_t^{t+\Delta t} \int_{CV} S dV dt,\end{aligned}$$

where the order of integration has been changed for the rate of change term. Using the unconditionally (numerically) stable backward Euler

(fully implicit) temporal discretization and M grid points over the spatial domain Ω , the system (9) can be written as a system of M equations on the form

$$a_P T_P = a_W T_W + a_E T_E + a_S T_S + a_N T_N + a_P^0 T_P^0 + S_u, \quad (11)$$

where the a 's are coefficients and T_P is the temperature at the grid point (point P) under consideration at time step $k + 1$. S_u and S_P arise from discretizing the source term S as $\Delta V \cdot S = S_u + S_P T_P$. Using the convenient compass notation, T_W , T_E , T_S and T_N are the temperatures at the west, east, south and north neighbor grid points, respectively, at time step $k + 1$. T_P^0 is the temperature at grid point P at time step k . Collecting the temperature at all grid points in a row vector $T(k) \in \mathbb{R}^M$ leads to a discrete linear system on descriptor form

$$\begin{aligned} ET(k+1) &= \bar{A}T(k) + \bar{B}u(k) + \bar{V}, \\ y(k) &= \bar{C}T(k), \end{aligned} \quad (12)$$

where $E \in \mathbb{R}^{M \times M}$ is a penta-diagonal matrix containing the coefficients a_p , a_W , a_E , a_S and a_N and $\bar{A} \in \mathbb{R}^{M \times M}$ is a diagonal matrix with a_P^0 on the main diagonal. $\bar{B} \in \mathbb{R}^{M \times m}$ contains the contributions from the inputs, while the constant source terms give rise to a constant term $\bar{V} \in \mathbb{R}^M$.

Linear systems such as (12) can be solved either by direct methods, such as matrix inversion or Gauss elimination, or by iterative methods, such as successive over-relaxation or more sophisticated solvers. In the case of non-linear systems, for instance when the thermal conductivity k is dependent on the temperature or the source term is non-linear, iteration is required. In the CFD community, iterative methods are often preferred rather than direct methods even when the equation system is linear, mainly due to significantly smaller storage requirements. In a domain comprising M grid points, a direct method will require the simultaneous storage of M^2 coefficients, while an iterative solver will only require the storage of the non-zero coefficients. Given the sparsity of the coefficient matrices, the profit is considerable.

To validate that the plate model is unstable, we rearrange system (12) into regular linear state space form by inverting E and multiplying throughout (12) to obtain

$$\begin{aligned} T(k+1) &= AT(k) + Bu(k) + V, \\ y(k) &= CT(k), \end{aligned} \quad (13)$$

where $A \in \mathbb{R}^{M \times M}$, $B \in \mathbb{R}^{M \times m}$, \mathbb{R}^M , $y \in \mathbb{R}^p$ and $C \in \mathbb{R}^{p \times M}$. The matrix E is indeed invertible in this case, since it is strictly diagonally dominant, and consequently non-singular. This is not necessarily possible in general, but it allows us to establish that the system matrix A has an unstable eigenvalue outside the unit circle,

$$|\lambda(A)|_{\max} = 1.00001 > 1. \quad (14)$$

3.2 Reduced order model

The PDE (9) is discretized using 50 grid points in both the x - and y -direction. This gives in total 2500 states in the full model (13). To construct a model of reduced order, (13) is simulated for 600 time steps, thus forming the matrix of snapshots \mathcal{X} . During this simulation the inputs are varied to excite as much of the dynamics of the system as possible. SVD of the snapshot matrix is performed, and the first four columns of Φ , corresponding to the four largest singular values are extracted to form the POD basis Φ_r , as depicted in figure 2. As can be seen from the figure the singular values fall off quite rapidly, and many of the singular values are zero, indicating that the basis functions corresponding to those singular values can be omitted without loss of information. With

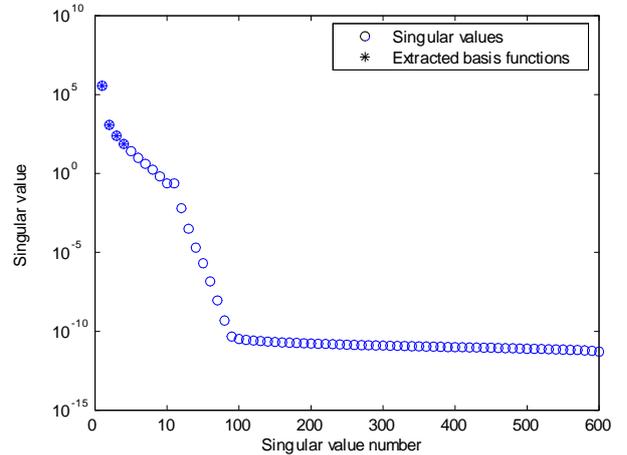


Figure 2: Singular values of the snapshot matrix. The *'s indicate singular values corresponding to the extracted basis functions. Note that the ordinate axis is logarithmic. From singular value 10-600, every tenth singular value is plotted, for clarity.

the first four basis functions, the heuristic crite-

rior

$$P = \frac{\sum_{i=1}^r \sigma_i^2}{\sum_{i=1}^M \sigma_i^2} \approx 1, \quad (15)$$

indicating a fairly accurate reduced order model [9]. The criterion gives an indication of the part of the energy that is conserved in the reduced order model. Moreover, if the reduced order state has dimension four, then the number of states in the reduced order model is equal to the number of inputs. Consequently, the reduced order model is fully actuated, which might be favorable when tracking a reference profile for the complete state. The basis Φ_r defines the linear dimensionality-reducing mapping $H : \mathbb{R}^M \mapsto \mathbb{R}^r$, such that $\alpha = H(T) = \Phi_r^T T$, and the expansion $G : \mathbb{R}^r \mapsto \mathbb{R}^M$ such that $T = G(\alpha) = \Phi_r \alpha$. To obtain the reduced order model on the form (8), the procedure presented in [9] is followed. By applying the inner product criterion (6) and inserting $T = \Phi \alpha$ into (12), the reduced order model can be obtained:

$$\Phi^T E \Phi_r \alpha(k+1) = \Phi^T A \Phi_r \alpha(k) + \Phi^T B u(k) + \Phi^T V \quad (16)$$

Defining $E_r \triangleq \Phi^T E \Phi_r$ allows us to write

$$\begin{aligned} \alpha(k+1) &= E_r^{-1} \Phi^T A \Phi_r \alpha(k) + E_r^{-1} \Phi^T B u(k) \\ &\quad + E_r^{-1} \Phi^T V, \end{aligned} \quad (17)$$

where E_r is invertible since E , Φ^T and Φ_r are all nonsingular. This yields the reduced-order model on discrete state space form

$$\begin{aligned} \alpha(k+1) &= \hat{A} \alpha(k) + \hat{B} u(k) + \hat{V}, \\ y(k) &= \hat{C} \alpha(k), \end{aligned} \quad (18)$$

where $\alpha \in \mathbb{R}^r$, $u \in \mathbb{R}^m$, $y \in \mathbb{R}^p$, $\hat{A} = E_r^{-1} \Phi^T A \Phi_r \in \mathbb{R}^{r \times r}$, $\hat{B} = E_r^{-1} \Phi^T B \in \mathbb{R}^{r \times m}$, $\hat{V} = E_r^{-1} \Phi^T V \in \mathbb{R}^r$ and $\hat{C} \in \mathbb{R}^{p \times r}$. In this example, $r = m = p = 4$, and \hat{C} is chosen to be $\mathbf{I}_{4 \times 4}$. In an actual implementation, α_i should be reconstructed from measurements on the real state $T(t, x, y)$ through an observer, rather than measuring the whole reduced state, as done in [9]. Here we merely assume that all states α_i are available for feedback.

The reduced order model is unstable, since

$$|\lambda(\hat{A})|_{\max} = 1.002 > 1, \quad (19)$$

that is, outside the unit circle. In fact, the unstable eigenvalue of the reduced-order model is larger than the unstable eigenvalue of the full model. Note that the general POD procedure does not automatically preserve stability properties during the model reduction process. Criteria for preserving stability properties are derived in [21].

4 Controller design

Feedback control is performed by use of heat flux actuators on parts of the boundary of the domain. The actuators are placed at the bold lines in figure 1. The control objective is to reach a constant temperature reference T_d while at the same time rejecting disturbances. The reference temperature is set to be a uniform temperature of 300°K . Since the full model is too large for controller design the reduced order model is analyzed instead. The reduced order state reference α_d is found as $\alpha_d = \Phi^T T_d$. Given the unstable reduced order model (18), the control objective is to stabilize the system around the reference temperature. Defining the tracking error $e(k) \triangleq \alpha_d - y(k)$, and recalling that $y(k) = \hat{C} \alpha(k) = \alpha(k)$, the control input is chosen as

$$u = Ke = -Ky(k) + K\alpha_d, \quad (20)$$

$$= -K\alpha(k) + K\alpha_d, \quad (21)$$

where K is chosen such that the eigenvalues of the closed-loop system matrix $(\hat{A} - \hat{B}K)$ have magnitude less than one.

Taking into consideration that the reduced-order model is merely an approximation, the controller should include integral action in order to minimize the steady-state error. To do this in a straightforward way, we define the augmented state

$$\tilde{\alpha}(k) \triangleq \begin{bmatrix} \alpha(k) \\ u(k-1) \end{bmatrix} \in \mathbb{R}^{r+m}, \quad (22)$$

giving an augmented state space model

$$\begin{aligned} \tilde{\alpha}(k+1) &= \tilde{A} \tilde{\alpha}(k) + \tilde{B} \Delta u(k) + \tilde{V}, \\ \tilde{y}(k) &= \tilde{C} \tilde{\alpha}(k), \end{aligned} \quad (23)$$

where

$$\begin{aligned} \tilde{A} &\triangleq \begin{bmatrix} \hat{A} & \hat{B} \\ 0 & I \end{bmatrix}, \quad \tilde{C} \triangleq [C \quad 0], \\ \tilde{B} &\triangleq \begin{bmatrix} \hat{B} \\ I \end{bmatrix}, \quad \tilde{V} \triangleq \begin{bmatrix} \hat{V} \\ 0 \end{bmatrix}, \end{aligned} \quad (24)$$

and $\Delta u(k) = u(k) - u(k-1)$. In this augmented state space, integral action is built-in, and the input increment $\Delta u(k)$ is found as

$$\Delta u(k) = K(\alpha_d - y(k)),$$

where K is the feedback gain matrix found above.

5 Numerical Simulation

Without control the temperature of the plate is strictly increasing. After 20 minutes the highest plate temperature is 368°K . If the simulation is run for a longer period of time the temperature continues to increase, illustrating the instability of the system. Now, the full CFD-model is simulated with the controller designed for the reduced order model in section 4. Initially, the plate temperature is equal to the ambient temperature at 293°K . At $t = 0$ the inner source and controller are switched on, and the system is simulated until steady-state occurs, after approximately 20 minutes. The largest steady-state error is approximately 4°K , as shown in figure 3.

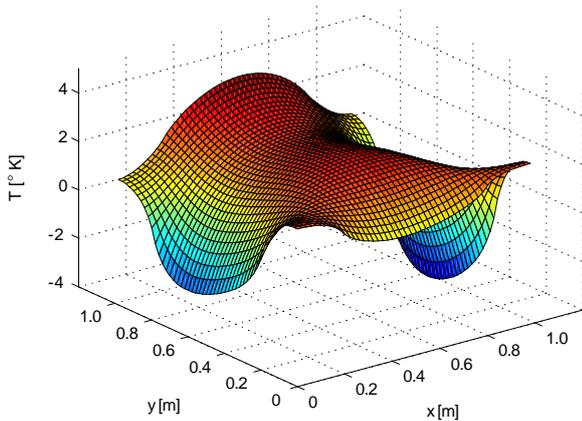


Figure 3: Steady-state error, after 20 minutes.

It is seen that although the original CFD model is symmetric, the controller based on the reduced-order model does not manage to exploit this symmetry, since the symmetry is not preserved in the model reduction scheme.

The simulation of the full closed-loop CFD model illustrates that the plant has been stabilized. The temperature distribution at steady-state is shown in figure 4.

6 Concluding Remarks

In this paper we have demonstrated, using a case study, that a CFD-model of an unstable system can be stabilized through model reduction and a controller designed for the reduced order model. This makes it possible to study stability proper-

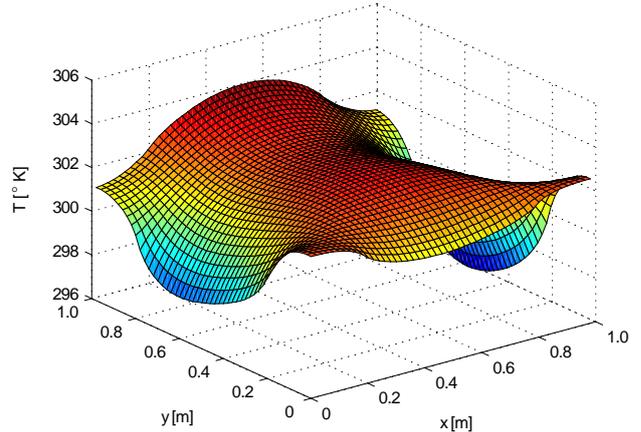


Figure 4: Steady-state temperature, after 20 minutes.

ties of systems that would otherwise be very computationally demanding.

Further and ongoing work include stability analysis of reduced-order models of more general PDE-models, and development of model reduction methodology. The 2D-plate model of the current work is viewed as a step on the way in this work.

References

- [1] Ole Morten Aamo and Miroslav Krstić. *Flow Control by Feedback: Stabilization and Mixing*. Springer, 2002.
- [2] Thomas R. Bewley. Flow control: new challenges for a new Renaissance. *Progress in Aerospace Sciences*, 37(1):21–58, 2001.
- [3] J. H. Ferziger and Milovan Peric. *Computational Methods for Fluid Dynamics*. Springer Verlag, 3rd edition, 2002.
- [4] J. D. Anderson. *Computational Fluid Dynamics*. McGraw-Hill International Editions, 1995.
- [5] H. K. Versteeg and W. Malalasekera. *An Introduction to Computational Fluid Dynamics*. Longman, 1995.
- [6] S. S. Ravindran. A reduced-order approach for optimal control of fluids using proper orthogonal decomposition. *International Journal for Numerical Methods in Fluids*, 34:425–448, 2000.
- [7] A.C. Antoulas, D.C. Sorensen, and S. Gugercin. A survey of model reduction methods for large-scale systems. V. *Olshesky (Ed.), Contemporary Mathematics*, 280:193–219, 2001.
- [8] S. S. Ravindran. Control of flow separation over a forward-facing step by Model reduction. *Computer Methods in Applied Mechanics and Engineering*, 191:4599–4617, 2002.
- [9] P. Astrid, L. Huisman, S. Weiland, and A. C. P. M. Backx. Reduction and predictive control design for

- a computational fluid dynamics model. In *Proc. 41st IEEE Conf. on Decision and Control*, volume 3, pages 3378–3383, Las Vegas, NV, 2002.
- [10] Patricia Astrid. *Reduction of process simulation models*. PhD thesis, Eindhoven University of Technology, 2004.
- [11] K. Karhunen. Zur spektral theorie stochastischer prozesse. *Ann Acad Sci Fennicae, Ser A1 Math Phys*, 34:1–7, 1946.
- [12] M. Loeve. Functiona aleatoire de second ordre. *Revue Science*, 84:195–206, 1946.
- [13] J. L. Lumley. The structure of inhomogeneous turbulent flow. In *Atmospheric Turbulence and Wave Propagation*, 1967.
- [14] Michael Kirby. *Geometric Data Analysis*. Wiley, 2001.
- [15] L. Sirovich. Turbulence and the dynamics of coherent structures. part 1: Coherent structures. *Quarterly of Applied Mathematics*, 45(3):561–571, 1987.
- [16] K. Kunisch and S. Volkwein. Control of the Burgers Equation by a Reduced-Order Approach Using Proper Orthogonal Decomposition. *Journal of Optimization Theory and Applications*, 102(2):345–371, August 1999.
- [17] G. Berkooz, P. Holmes, and J. Lumley. The proper orthogonal decomposition in the analysis of turbulent flows. *Annual Review of Fluid Mechanics*, 25:539–575, 1993.
- [18] S. Volkwein. Proper orthogonal decomposition and singular value decomposition. Technical Report 103, SFB-Preprint, 1999.
- [19] C. Canuto, M. Y. Hussaini, A. Querteroni, , and T. A. Zang. *Spectral Methods in Fluid Dynamics*. Springer Series in Computational Physics, 1988.
- [20] P. Constantin, C. Foias, B. Nicolaenko, and R. Temam. *Integral Manifolds and Inertial Manifolds for Dissipative Partial Differential Equations*. Springer Verlag, 1989.
- [21] Stephen Prajna. POD Model Reduction with Stability Guarantee. In *Proc. 42nd IEEE Conf. on Decision and Control*, Hawaii, 2003.